Amendments to the Claims:

This listing of claims will replace all prior versions, and listings, of claims in the application:

Listing of Claims:

1. **(Currently Amended)** A method for operating an air conditioning system of a vehicle, comprising:

circulating, in a circuit, a fluid to condition an airstream;

operating the circuit in a cooling mode or a heating mode, wherein**, in the heating mode,** the circuit includes a condenser or a compressor, a heat exchanger, and an intermediate store ~~**in the heating mode**~~; **[[and]]**

circulating, in an evaporator, the fluid; and

controlling the circuit, such that the intake pressure of the condenser or the compressor at least partially overshoots a saturation pressure in the circuit caused by the ambient temperature**, by having a volume of the evaporator smaller than a storage volume of the intermediate store**.

2. **(Previously Presented)** The method as claimed in claim 1, further comprising operating the circuit during the heating mode in a dextrorotary triangulation process, wherein an exit side of the condenser or the compressor connects to an entry side of a control valve, an exit side of the control valve connects to an entry side of the heat exchanger, an exit side of the heat exchanger connects to an entry side of the intermediate store, and an exit side of the intermediate store connects to an entry side of the condenser or the compressor.

3. **(Previously Presented)** The method as claimed in claim 1, further comprising controlling intake pressure in a range of 10 bar to 110 bar.

4. **(Previously Presented)** The method as claimed in claim 1, further comprising dividing the fluid in the circuit into at least one active part and at least one passive part while in the heating mode.

5. **(Previously Presented)** The method as claimed in claim 1, further comprising:

activating the heating mode; and

routing the fluid out of the passive part of the circuit into the active part of the circuit.

6. **(Previously Presented)** The method as claimed in claim 1, further comprising routing out the fluid of the passive part of the circuit into the active part of the circuit when a predeterminable threshold value for the intake pressure in the active part of the circuit is undershot.

7. **(Previously Presented)** The method as claimed in either claim 5, further comprising transferring the fluid out of the passive part of the circuit into the active part of the circuit by changing the circuit operated in the heating mode over to the cooling mode.

8. **(Previously Presented)** The method as claimed in either claim 5, further comprising transferring the fluid out of the passive part of the circuit into the active part of the circuit by changing the circuit operated in the heating mode to a laevorotatory triangular process.

9. **(Previously Presented)** The method as claimed in either claim 7, further comprising:

operating the circuit in the cooling mode or in the laevorotatory triangulation process up to the undershooting of a settable threshold value; and

changing the circuit to the heating mode again after the undershooting of the threshold value.

10. **(Previously Presented)** The method as claimed in claim 9, further comprising predetermining the threshold value for an intake pressure and/or for a high pressure and/or for a hot-gas temperature at the condenser or the compressor.

11. **(Previously Presented)** The method as claimed in claim 9, further comprising setting the threshold value of the intake pressure at at least 3 bar below the value of the saturation pressure caused by the ambient temperature.

12. **(Previously Presented)** The method as claimed in claim 7, further comprising:

operating the circuit in the cooling mode or in the laevorotatory triangulation process for a predeterminable period of time; and

changing the circuit to the heating mode after expiration of the period of time.

13. **(Previously Presented)** The method as claimed in claim 7, further comprising reducing an air stream through the evaporator after the changeover to the cooling mode or to the laevorotatory triangulation process.

14. **(Previously Presented)** The method as claimed in claim 7, further comprising reducing an air stream through a gas cooler after the changeover to the cooling mode or to the laevorotatory triangulation process.

15. **(Previously Presented)** The method as claimed in claim 10, further comprising equalizing pressure in the circuit after returning to the heating mode.

16. **(Currently Amended)** An air conditioning system for a vehicle comprising:

a circuit, operable in a cooling or heating mode, configured to circulate a fluid, the fluid configured to condition an air stream, wherein in the heating mode, the circuit includes: **(a)** a heat exchanger, **(b)** an intermediate store, and **(c)** a condenser or a compressor for the intermediate storage or for the condensation of the fluid**[[,]]; and**

an evaporator, connected to the circuit, for fluid reception,

wherein **a volume of the evaporator is smaller than a storage volume of the intermediate store, such that** the condenser or the compressor is configured to operate at an intake pressure, that is higher than the saturation pressure in the circuit, caused by the ambient temperature.

17. **(Currently Amended)** The air conditioning system as claimed in claim 16, ~~**further comprising:**~~

~~**an evaporator,**~~

wherein a secondary side **of the evaporator** is included in a flow duct of the air stream **[[on]]**, a primary side **of the evaporator** is connected to the circuit, and an exit side **of the evaporator** is connected to the intermediate store, and

wherein a nonreturn valve is interposed between the evaporator and the intermediate store.

18. **(Cancelled)**.

19. **(Currently Amended)** The air conditioning system as claimed in claim **[[18]] 16**, wherein the ratio of the storage volume of the intermediate store to the volume of the evaporator lies in the range of 2:1 to 20:1.

20. **(Previously Presented)** The air conditioning system as claimed in claim 16, further comprising a control device arranged between the heat exchanger and the intermediate store.

21. **(Previously Presented)** The air conditioning system as claimed in claim 16, further comprising a pressure sensor assigned on the intake side to the condenser or the compressor.

22. **(Previously Presented)** The air conditioning system as claimed in claim 16, wherein the circuit is subdivided into at least one active part and at least one passive part.

23. **(Previously Presented)** The air conditioning system as claimed in claim 22, wherein the active part is connected to the passive part by a control device configured to open when the fluid quantity in the active part of the circuit overshoots a predeterminable threshold value.

24. **(Previously Presented)** The air conditioning system as claimed in claim 19, wherein the condenser or the compressor is connected to the evaporator on the exit side via a control means and on the entry side via an associated controllable connecting line after the opening of the control means gaseous fluid passes into the evaporator and forces liquid fluid out of the evaporator into the active part of the circuit.

25. **(Previously Presented)** The method as claimed in claim 11, further comprising setting the threshold value of the intake pressure at 5 bar below the value of the saturation pressure caused by the ambient temperature.

26. **(Previously Presented)** The air conditioning system as claimed in claim 19, wherein the ratio of the storage volume of the intermediate store to the volume of the evaporator lies in the range of between 2:1 and 10:1.